

TABLE 1. AVERAGE MAXIMUM WATER MAIN TEMPERATURE^a (Concluded)

STATE	CITY	TEMP. F	STATE	CITY	TEMP. F
Okla.	Oklahoma City	82	Utah	Logan	44
	Tulsa	85		Salt Lake City	60
Ore.	Eugene	60	Va.	Fredericksburg	75
	Portland	64		Lynchburg	73
Pa.	Altoona	74		Norfolk	80
	Erie	75	Wash.	Olympia	58
	Johnstown	74		Seattle	62
	McKeesport	82		Spokane	51
	Philadelphia	83		Tacoma	57
	Pittsburgh	81	W. Va.	Charleston	85
R. I.	Providence	68		Huntington	78
S. C.	Charleston	80		Wheeling	78
	Greenville	81	Wis.	LaCrosse	54
	Spartanburg	78		Madison	58
S. Dak.	Rapid City	55		Milwaukee	70
Tenn.	Chattanooga	84		Racine	68
	Knoxville	89			
	Memphis	70			
	Nashville	90			
Texas	Amarillo	65	PROVINCE		
	Austin	90			
	Beaumont	86			
	Dallas	86	Alta.	Calgary	64
	Fort Worth	84	B. C.	Vancouver	60
	Galveston	90	Ont.	London	50
	Houston	84		Toronto	63
	Port Arthur	83	P. E. I.	Charlottetown	48
	San Antonio	76	Que.	Montreal	78
	Wichita Falls	85		Quebec	68

^aThese averages taken from various city water main locations, with some actual values slightly higher and some lower than values shown.

As the heat content of the air increases, its wet-bulb temperature rises. (See Chapter 1.) Because it is impractical to leave the air in contact with water for a long enough time to permit the wet-bulb temperature of the air and the temperature of the water to reach equilibrium, atmospheric water cooling equipment aims to circulate only enough air to cool the water to the desired temperature with the least possible expenditure of power.

In an air washer, humidifier or dehumidifier, the air is first conditioned by water to change its moisture and temperature, and it is then sent to the place where it is to be used. In water cooling equipment the temperature of the water is reduced by air, and the cooled water is carried to its point of usage. In the air washer, an excess of water is used to condition a fixed quantity of air, while in water cooling equipment, an excess of air is used to cool a fixed quantity of water.

Both types of equipment have a common basis of design, however, in that the size of the equipment is determined by the quantity of air that must be handled. With the air washer, the size of the equipment is fixed by the quantity of air to be conditioned, and the amount of conditioning is controlled by the quantity and temperature of the water supplied and its method of application. With water cooling apparatus, its size and the

quantity of air required bear no direct relation to the quantity of water being cooled, but vary through a wide range for different services and conditions.

Sizes of Equipment

Assuming a definite quantity of water to be cooled, the size and design of atmospheric cooling equipment are affected by the following factors:

1. Temperature range through which the water must be cooled.
2. Number of degrees above the wet-bulb temperature of the entering air to which the water temperature must be reduced.
3. Temperature of the atmospheric wet-bulb at which the required cooling must be performed.
4. Time of contact of the air with the water. (This involves height or length of the apparatus and velocity of air.)
5. Surface of water exposed to each unit quantity of air.
6. Relative velocity of air and water.

Items 1, 2, and 3 are established by the type of service and geographical location, while items 4, 5, and 6 depend upon the design of the equipment.

The establishment of a proper cooling range depends upon:

1. Type of service (refrigerating, internal combustion engine and steam condensing).
2. Wet-bulb temperature at which the equipment must operate satisfactorily.
3. Type of condenser or heat-exchanger used.

Because the design of an entire plant is usually affected by the quantity and temperature of the cooling water supply, plants should be designed for cooling water conditions which can be most efficiently attained. The first consideration is usually the limiting temperature of the plant. For example, if an ammonia compressor refrigerating plant is to be designed for 185 lb head pressure as a normal maximum, the limiting temperature of the ammonia in the condenser is 96 F. Should the ammonia temperature go above this figure the head pressure will exceed 185 lb and power consumption increases. To obtain this head pressure, the temperature of the circulating water leaving the condenser must always be less than 96 F by an amount depending upon the size and design of the condenser, the

TABLE 2. CONDENSER DESIGN DATA

GAS	MAXIMUM PRESSURE DESIRED IN CONDENSER	GAS TEMPERATURE IN CONDENSER DEG F	LEAVING HOT WATER TEMPERATURE DEG F	
			Best Condenser Design	Average Condenser Design
Steam	28 in. vacuum	101.2	97	93
Steam	27 in. vacuum	115.1	110	105
Steam	26 in. vacuum	125.9	120	114
Ammonia	185 lb gage head pressure	96.0	92	88
Carbon dioxide	1030 lb gage head pressure	86.0	83	81
Methyl chloride	102 lb gage head pressure	100.0	96	92
Dichlorodifluoromethane	117 lb gage head pressure	100.0	96	93