

of required air-flow rates cannot apply. Furthermore, it would not be economical to increase the air-flow rate through such a local exhaust hood to bring it up to general ventilation volumes because the suction power requirements would be exorbitant. General ventilation should be provided by low-velocity, low-suction equipment. The warning, then, is not to overestimate the capability of an exhaust hood as a general ventilating device. Special systems of multiple hoods operating simultaneously in small rooms have been observed in some cases to give high rates of air change for the space. This condition is not a property of exhaust hoods, but instead the result of greatly reduced air space per hood.

1. Exhausting vs. Blowing

Figure 4 illustrates the fundamental difference between blowing and exhausting devices. Air blown from a small opening retains its directional effect for a considerable distance beyond the plane of the opening. However, if the flow of air is

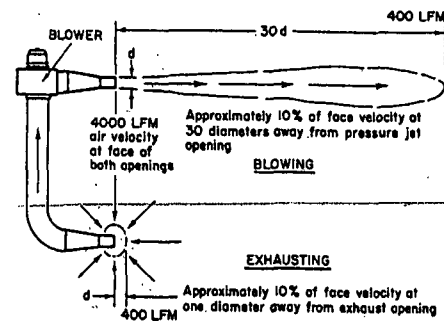


Figure 4. Air-flow characteristics of exhausting and blowing devices.

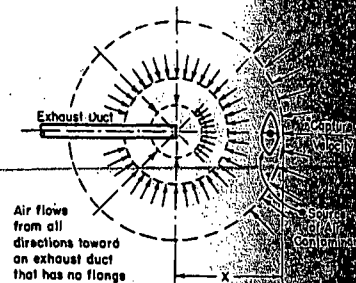


Figure 5. Spherical nature of air flow into an unflanged, unobstructed exhaust hood.

reversed so that the same opening (or its equivalent in size) now handles the same volume of air by exhaust, the flow becomes almost completely nondirectional and its range-of-influence is greatly reduced. It is for this reason that local exhaust hoods should not be contemplated for any process that cannot be conducted in the immediate vicinity of the hood.

2. DallaValle's Equation for Unobstructed Hoods

This equation is used to determine the relation between the quantity of air flowing into an exhaust hood and the resulting air velocity at a distant point along the hood centerline. The equation applies only when there is no obstruction between the hood opening and the control point in question (see Figure 5).

A simplified form of the equation for both round and rectangular hood openings is as follows:

$$Q = V \left(\frac{X^2}{b} + A \right)$$

where A = area of hood opening in square feet, b = a constant that depends upon the shape of the hood, Q = quantity of air flow in c.f.m., V = centerline air velocity in f.p.m. at X feet from opening, and X = distance in feet from hood opening to control point.

DallaValle's equation is used frequently with a value of 0.1 for constant b , giving a simple approximate formula suitable for round or essentially square hoods. With this constant the equation is given in the convenient form: $Q = 1.405 V \left(\frac{X^2}{b} + A \right)$. Table 7 lists the constants and equations for hood openings of various proportions.

3. Flanges and Baffles Increase the Range of Exhaust Hoods

Figure 6 illustrates the beneficial effect of restricting the direction of air flow into an exhaust opening. The diagrams were constructed on the basis of a point source of exhaust which, of course, does not occur in practice. Figure 7 shows the

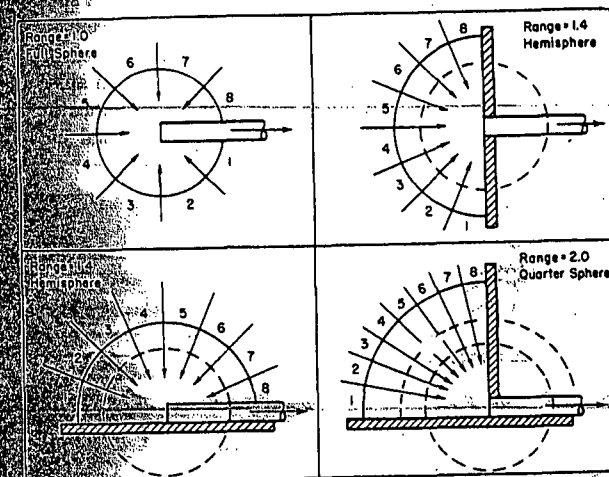


Figure 6. Flanges and baffles increase the range of exhaust openings.

Figure 7 shows the velocity contours and streamlines for a rectangular hood and indicates the marked deviation from spherical contours at points close to the hood face. Figure 8 illustrates a flanged "spot exhaust" hood for arc welding.

W. N. DallaValle, *Exhaust Hoods*. Industrial Press, New York, 1944.