

(Refer to Chapter 35 for coil selection and efficiency.)

From note under Equation 19 the dry-bulb range will be $(80 - 54.6) \times 0.85 = 21.6$ deg, and the dry-bulb temperature of air leaving the coil will be $80 - 21.6 = 58.4$ F. The dry-bulb temperature leaving the fan (including the heat supplied by the fan motor), or delivered into the room, will be (from Equation 21):

$$t_a = 80 - \frac{184,415 - 19,100}{1.08 \times 7900} = 80 - 19.4 = 60.6 \text{ F.}$$

With good distribution and diffusion, this temperature should not produce objectionable drafts.

EXAMPLE 18: SUMMARY

Outdoor Conditions.....	95 DB	78 WB	0.0169 Humidity Ratio
Space Conditions.....	80 DB	65 WB	0.0098 Humidity Ratio
DIFFERENCE.....	15		0.0071

SENSIBLE LOAD

Transmission	Btu/Hr
Roof 4000 sq ft $\times 53^\circ \times 0.34 =$	72,000
S. Wall 405 sq ft $\times 6^\circ \times 0.41 =$	995
E. Wall 765 sq ft $\times 11^\circ \times 0.52 =$	4,380
N. Wall Ex. 170 sq ft $\times 3^\circ \times 0.52 =$	265
N. & W. Party Wall 1065 sq ft $\times 2^\circ \times 0.26 =$	550
Floor None	
Door 35 sq ft $\times 15^\circ \times 0.59 =$	310
All Glass and Rest of Doors =	3,020

Solar Radiation

S. Glass 60 sq ft $\times 14 =$	840
S. Glass (Doors) 35 sq ft $\times 42 =$	1,470
E. Glass (Doors) 18 sq ft $\times 14 =$	250
N. Glass 30 sq ft $\times 15 =$	450

Internal Load

Infiltration 67 cfm $\times 1.08 \times 15^\circ =$	1,085
Lights $(12,000 \times 1.20 + 4000) 3.41 =$	62,700
People 85 $\times 200 =$	17,000
Motor, Fan 7.5 hp $\times 2544 =$	19,100

TOTAL SENSIBLE SPACE LOAD..... 184,415

LATENT LOAD

Infiltration 67 cfm $\times 4840 \times 0.0071 =$	2,300
People 85 $\times 250 =$	21,250

TOTAL LATENT SPACE LOAD..... 23,550

VENTILATION AIR

Sensible 1275 cfm $\times 1.08 \times 15^\circ =$	20,700
Latent 1275 cfm $\times 4840 \times 0.0071 =$	43,800

GRAND TOTAL LOAD..... 272,465

LETTER SYMBOLS USED IN CHAPTER 12

- α = fraction of incident solar radiation absorbed, dimensionless; subscripts D, d, t refer to direct, diffuse and total, respectively.
- β = solar altitude, degrees.
- γ = wall solar azimuth, degrees.
- ϵ = emissivity, dimensionless.
- θ = incident angle, degrees.
- λ = amplitude decrement factor, dimensionless.
- μ = permeability to moisture transmission, grains per (square foot) (hour) (inch mercury).

Cooling Load

- τ = fraction of incident solar radiation transmitted, dimensionless.
Subscripts D, d, t refer to direct, diffuse and total, respectively.
- ϕ = solar azimuth, degrees.
- ψ = wall azimuth, degrees.
- A = area across which heat is being transferred, square feet.
- f = unit surface conductance, Btu per (hour) (square foot) (Fahrenheit degree).
Subscripts c, r, o, and i refer to convection, radiation, outdoor, and indoor, respectively.
- G_T = fraction of total window area receiving direct solar radiation when shaded by window reveal, dimensionless.
- h = enthalpy of air per pound of dry air, Btu per pound.
Subscripts i, o, and s refer to indoor, outdoor, and supply air, respectively.
- I = incident solar radiation, Btu per (hour) (square foot).
Subscripts D, d, Dn, and t refer to direct, diffuse, direct normal and total solar radiation, respectively.
- K = cosine of angle of incidence for direct solar radiation striking a surface, dimensionless.
- k = thermal conductivity of building material, Btu per (square foot) (hour) (Fahrenheit degree per inch).
- L = thickness of building material, inches.
- l = height of window, feet.
- Q = rate of entry of outdoor air, cubic feet per minute.
- Q_a = required air quantity through conditioning equipment, cubic feet per minute.
- q = instantaneous rate of heat transfer, Btu per hour.
- q_a = instantaneous latent heat load, Btu per hour.
- q_m = latent heat load due to moisture transmission through materials, Btu per (hr) (sq ft).
- q_s = instantaneous sensible heat load, Btu per hour.
- q_t = $q_s + q_a$, Btu per hour.
- R_o = low temperature radiant energy received from outdoor surroundings (does not include solar radiation), Btu per (hr) (sq ft of receiving surface).
- R = radiant energy emitted by a black body, Btu per (hr) (sq ft).
Subscripts go and L refer to outdoor surfaces of glass and building, respectively.
- S = rate of heat storage within a glass section, Btu per (hr) (sq ft).
- t_a = sol-air temperature, Fahrenheit degrees.
- t_a^* = sol-air temperature at a time earlier than the time for which heat gain is being found by an amount that is equal to the time lag of the wall or roof, Fahrenheit degrees.
- t_{a1} = temperature of indoor glass surface, Fahrenheit degrees.
- t_{a2} = temperature of outdoor glass surface, Fahrenheit degrees.
- t_1 = indoor air temperature, Fahrenheit degrees.
- t_o = temperature of outer surface of building, Fahrenheit degrees.
- t_m = 24-hr cyclic average sol-air temperature, Fahrenheit degrees.
- t_o = outdoor air temperature, Fahrenheit degrees.
- t_p = $t_m + (t_a^* - t_m)$, net equivalent outdoor temperature for combined periodic and mean heat flow, Fahrenheit degrees.
- t_r = room supply air dry-bulb temperature, Fahrenheit degrees.
- U = overall coefficient of heat transfer of a structural section, Btu per (square foot) (hour) (Fahrenheit degree).
- v_o = volume of outdoor air per pound of dry air, cubic feet.
- w = width of window, feet.
- W = humidity ratio, pounds moisture per pound of dry air.
Subscripts i, o, and s refer to indoor, outdoor, and supply air, respectively.